## APPENDIX A

 DESIGN OF FENDERS
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## A. 1 Example 1

## Given

Details of design vessels for a proposed solid pier are given as follows :

|  | Type 1 Vessel | Type 2 Vessel |
| :--- | :---: | :---: |
| Length | 8.0 m | 35.0 m |
| Beam | 5.0 m | 20.0 m |
| Depth | 1.2 m | 2.5 m |
| Draft | 0.7 m | 1.5 m |
| Displacement | 25 tonnes | 400 tonnes |
| Bent radius of bow side of vessel | 10.0 m | 25.0 m |

Determine the layout and the size of rubber fenders.

## Solution

(a) Length of fenders

Low water level $=+0.2 \mathrm{mPD}$
High water level $=+3.0 \mathrm{mPD}$
Freeboard for Type 1 Vessel $=1.2-0.7=0.5 \mathrm{~m}$
Freeboard for Type 2 Vessel $=2.5-1.5=1.0 \mathrm{~m}$
Length of fenders $=3.5 \mathrm{~m}$, extending from +0.15 mPD to +3.65 mPD to allow berthing under low and high water levels.

The relationship between fenders and vessels at different water levels is shown as follows :


## (b) Spacing of fenders (See Figure 8)

The length of the small vessel (Type 1 Vessel) using the pier $=8.0 \mathrm{~m}$
Maximum fender spacing $=0.15 \times 8.0 \mathrm{~m}=1.2 \mathrm{~m}$

Smallest bent radius of bow side of vessel $=10.0 \mathrm{~m}$
Height of fenders $=0.25 \mathrm{~m}$
Maximum fender spacing $=2 \times\left[10^{2}-(10-0.25)^{2}\right]^{1 / 2}=4.4 \mathrm{~m}$

Therefore, use fender spacing of 1.2 m .
(c) Size of fenders

Based on the dimensions of the vessels :

- Distance of the point of contact of Type 1 Vessel from its centre of mass $=3.0 \mathrm{~m}$
- Distance of the point of contact of Type 2 Vessel from its centre of mass $=12.0 \mathrm{~m}$

Assume that the angle between the line joining the point of contact to the centre of mass and the velocity vector of the vessel $=45^{\circ}$

Calculation of berthing energy (see Section 5.12 of Part 1 of the Manual) :

|  | Type 1 Vessel | Type 2 Vessel |
| :---: | :---: | :---: |
| $\mathrm{V}_{\mathrm{b}}$ | $0.4 \mathrm{~m} / \mathrm{s}$ | $0.3 \mathrm{~m} / \mathrm{s}$ |
| $\mathrm{C}_{\mathrm{m}}=1+2\left(\mathrm{D}_{\mathrm{v}} / \mathrm{B}_{\mathrm{v}}\right)$ | $1+2(0.7 / 5.0)=1.28$ | $1+2(1.5 / 20.0)=1.15$ |
| $\begin{aligned} & \mathrm{C}_{\mathrm{e}}=\left(\mathrm{K}_{\mathrm{v}}{ }^{2}+\mathrm{R}_{\mathrm{v}}{ } \cos ^{2} \mathrm{v}\right) /\left(\mathrm{K}_{\mathrm{v}}{ }^{+} \mathrm{R}_{\mathrm{v}}{ }^{2}\right) \\ & \text { Where } \mathrm{K}_{\mathrm{v}}=\left(0.19 \mathrm{C}_{\mathrm{b}}+0.11\right) \mathrm{L}_{\mathrm{v}} \end{aligned}$ | $\begin{aligned} \mathrm{K}_{\mathrm{v}}= & {\left[0.19 \times 25 \times 10^{3} /(8\right.} \\ & \times 5 \times 0.7 \times 1025) \\ & +0.11] \times 8=2.2 \\ \mathrm{C}_{\mathrm{e}}= & \left(2.2^{2}+3.0^{2} \cos ^{2} 45\right) \\ & /\left(2.2^{2}+3.0^{2}\right)=0.67 \end{aligned}$ | $\begin{aligned} \mathrm{K}_{\mathrm{v}}= & {\left[0.19 \times 400 \times 10^{3} /(35\right.} \\ & \times 20 \times 1.5 \times 1025) \\ & +0.11] \times 35=6.32 \\ \mathrm{C}_{\mathrm{e}}= & \left(6.32^{2}+12.0^{2} \cos ^{2} 45\right) \\ & /\left(6.32^{2}+12.0^{2}\right)=0.61 \end{aligned}$ |
| $\mathrm{C}_{\text {s }}$ | 1.0 | 1.0 |
| $\mathrm{C}_{\mathrm{c}}$ (for solid pier) | 0.9 | 0.9 |
| Berthing energy | $\begin{gathered} 0.5\left(1.28 \times 25 \times 0.4^{2} \times 0.67 \times\right. \\ 1.0 \times 0.9)=1.5 \mathrm{kNm} \end{gathered}$ | $\begin{gathered} 0.5\left(1.15 \times 400 \times 0.3^{2} \times 0.61 \times\right. \\ 1.0 \times 0.9)=11.4 \mathrm{kNm} \end{gathered}$ |

The total energy to be absorbed for accidental loading should be at least $50 \%$ greater than that for normal loading.

Therefore, select a rubber fender from the supplier's catalogue with design energy absorption
capacity greater than $11.4 \times 1.5=17.1 \mathrm{kNm}$

Since the pier has many fenders installed at a close spacing, the effect of angular compression on the fenders is neglected.

From the performance curve of the fender, the berthing reaction $=325 \mathrm{kN}$

## Performance curve of fenders



## A. 2 Example 2

## Given

The existing timber fenders of a piled deck ferry pier are to be replaced by plastic fenders. Determine the size of the plastic fenders, waling and rubber buffers. The design data are as follows:

Design vessel
Displacement $=940$ tonnes
Length $\quad=65.0 \mathrm{~m}$
Beam $\quad=12.0 \mathrm{~m}$
Depth $\quad=4.3 \mathrm{~m}$
Draft $\quad=2.0 \mathrm{~m}$

## Existing fender frame



Dimensions in mm

## Solution

(1) Try the following dimensions for the components of the fender frame

Plastic fender
Size $\quad=300 \mathrm{~mm} \times 300 \mathrm{~mm}$
Modulus of elasticity, $\mathrm{E}=32 \mathrm{Mpa}$
Moment of inertia, $\mathrm{I} \quad=0.0098 \mathrm{~m}^{4}$
Bending stress, $\sigma \quad=7 \mathrm{Mpa}$
Allowable moment $\quad=\sigma \mathrm{I} / \mathrm{y}=7 \times 10^{6} \times 0.0098 /(0.3 / 2)=457.3 \mathrm{kNm}$

## Rubber buffer

Cylindrical fender with outside diameter $=500 \mathrm{~mm}$

## Steel waling

Section $\quad=356 \times 406 \times 340 \mathrm{~kg} / \mathrm{m}$ Universal Column (Grade 43)
Modulus of elasticity, $\mathrm{E}=200 \mathrm{MPa}$
Design stress $\quad=265 \mathrm{Mpa}$
Plastic Modulus, $\mathrm{S} \quad=6.03 \times 10^{6} \mathrm{~mm}^{3}$
Moment capacity $\quad=265 \times 6.03=1598 \mathrm{kNm}$
(2) Calculation of berthing energy

Assume the distance of the point of contact of the vessel from its centre of mass $=20 \mathrm{~m}$

Angle between the line joining the point of contact to the centre of mass and the velocity vector, $\gamma=45^{\circ}$
$\mathrm{V}_{\mathrm{b}}=0.3 \mathrm{~m} / \mathrm{s}$
$\mathrm{C}_{\mathrm{m}}=1+2(2.0 / 12)=1.33$
$\mathrm{K}_{\mathrm{v}}=\left[0.19 \times 940 \times 10^{3} /(65 \times 12 \times 2.0 \times 1025)+0.11\right] \times 65=14.41$
$\mathrm{C}_{\mathrm{e}}=\left(14.41^{2}+20^{2} \cos ^{2} 45\right) /\left(14.41^{2}+20^{2}\right)=0.67$
$\mathrm{C}_{\mathrm{s}}=1.0$
$\mathrm{C}_{\mathrm{c}}=1.0$ for piled deck pier

Berthing energy $=0.5 \times\left(1.33 \times 940 \times 0.3^{2} \times 0.67 \times 1.0 \times 1.0\right)=37.7 \mathrm{kNm}$
(3) Capacity of the fender frame

The capacity of the fender system is checked for the following load cases:

Load case 1: Berthing load at close proximity to the buffer
A

$\stackrel{375}{\longleftrightarrow} \quad 5 @ 650=3250 \longrightarrow \underset{\longleftrightarrow}{375}$

Let the berthing load be P .
Assume that the design vessel collides with two fenders at one time and the berthing load P is equally shared between the two fenders. Since the berthing load $P$ is very close to the buffer, it is assumed that all the berthing energy is absorbed by the buffer at B only. The total energy to be absorbed for accidental loading should be at least $50 \%$ greater than that for normal loading.

Berthing energy to be absorbed $=37.7 \times 1.5=56.6 \mathrm{kNm}$

## Performance curve of buffers



From the performance curve of the buffer,
Reaction force $=590 \mathrm{kN}$
Deflection $=48 \% \times 500=240 \mathrm{~mm}<300 \mathrm{~mm}$


Load case 2: Berthing load at mid-span of steel waling


Assume that the total berthing energy is to be equally shared between two buffers ( A and B ). The total energy to be absorbed for accident loading should be at least $50 \%$ greater than that for normal loading.
Therefore, berthing energy to be absorbed by one buffer $=37.7 \times 1.5 / 2=28.3 \mathrm{kNm}$

From the performance curve of the buffer, the reaction force at $\mathrm{B}=600 \mathrm{kN}$
By symmetry, the reaction force at $\mathrm{A}=600 \mathrm{kN}$ and $\mathrm{P} / 2=600 \mathrm{kN}$

Checking moment capacity of steel waling :
Maximum Moment in steel waling $=600 \times 2-600 \times 0.375=975 \mathrm{kN}$
Taking a load factor of 1.6 according to BS 5950
Factored moment $=1.6 \times 975=1560 \mathrm{kNm}(<1598 \mathrm{kNm}, \mathrm{OK})$

Checking deflection of steel waling
Allowable deflection $=\mathrm{L} / 360=4000 / 360=11.1 \mathrm{~mm}$
Actual deflection of steel waling at mid-span $=6.1 \mathrm{~mm}(<11.1 \mathrm{~mm}, \mathrm{OK})$

Note: In principle, when the load is applied at the steel waling, there is some energy absorbed by the steel waling due to bending. When the load is at mid-span, this energy is approximately equal to $3.7 \mathrm{kNm}(1 / 2 \times 0.0061 \times 600+1 / 2 \times 0.0061 \times 600)$. It is negligible in comparison to the total berthing energy of 56.5 kNm .

Load case 3: Berthing load at mid-span of plastic fenders


Similarly to load case 2 , assume that the total berthing energy is equally shared between two buffers ( B and D ). Therefore, the reaction force at $\mathrm{B}=600 \mathrm{kN}$ By symmetry, the reaction force at $\mathrm{A}=600 \mathrm{kN}$ and $\mathrm{P} / 2=600 \mathrm{kN}$

Deflection of a plastic fender at mid-span
$=\mathrm{FL}^{3} / 16 \mathrm{EI}$
(where F is the applied load at mid-span)
$=(\mathrm{P} / 2)(3)^{3} /\left(16 \times 32 \times 10^{6} \times 0.0098\right)$
$=3.3 \mathrm{~mm}$

Maximum moment in one plastic fender
$=(\mathrm{P} / 2)(3) / 4$
$=450 \mathrm{kNm}(<457.3, \mathrm{OK})$

Note : In principle, when the load is applied at the two plastic fenders, there is some energy absorbed by the plastic fenders due to bending. When the load is at mid-span, this energy is approximately equal $2.0 \mathrm{kNm}(1 / 2 \times 0.0033 \times 600+1 / 2 \times 0.0033 \times 600)$. This is negligible in comparison to the total berthing energy of 56.5 kNm .

